

Problem (1):

(25 Marks)

- I)- Use the finite element technique to find
(a) the nodal displacements, strain and stress induced in the axially loaded stepped bar shown in Fig.1-a
(b) Formulate the eigen-value problem of the system.
 $A_3=2A_2=4A_1=16\text{cm}^2$. $\rho_1=\rho_2=\rho_3=7.8 \times 10^3 \text{ kg/m}^3$
II) Use a three element model to approximate the lowest natural frequency and mode shape for the system shown in Fig.1-b .

Problem (2):

(25 Marks)

- The milling machine shown in Fig.2 is mounted on spring-damper set and the machine tool is connected to an isolator . The driving motor of the machine vibrates due to the out of the balance of its rotor which is equivalent to a force : $F_m=1200\sin 90t \text{ N}$. The machine and the machine tool are characterize by :
Machine as : $M_m = 400 \text{ kg}$, $K_m=106 \text{ N/m}$, $C_m = 8000 \text{ N.s/m}$
Tool as : $m_t = 1 \text{ Kg}$, $K_t=3600 \text{ N/m}$, $C_t= 21.4 \text{ N.s/m}$. Determine :
1- the dynamic amplitude of the milling machine (X_{sm}).
2- The amplitude of the chatter (U) and the amplitude of the machine tool (X_t).
3- The proper damped absorber (m_a, K_a, C_a) such that the permissible chatter $U_{per} = 0.5 \text{ mm}$.

Problem (3):

(25 Marks)

A machine is modeled as 2-DOF system as shown in Fig.3 and characterized by : $m_1 = 0.5 \text{ m}_2 = 5 \text{ kg}$, $K_1=K_2=0.5 K_3 = 20000 \text{ N/m}$
 $C_1=4 C_2=0.5 C_3= 80 \text{ N.S/m}$

- 1- construct a model as a single degree of freedom system
- 2- prove that the damping in the system is proportional damping, and
- 3- calculate the pole location , residue , modal parameter model and hence

Problem (4)

(25 Marks)

- a- A cantilever of flexural rigidity EI , length L and mass per unit length ρ performs a transverse vibration . If the free end of the beam is fastened to a motor of mass m as shown in Fig.4 derive the frequency equation of the present continuous system.
b- A rotor of mass 10 kg and unbalance $m_0.e=0.01 \text{ kg.m}$ of speed $\Omega=5 \text{ sec}^{-1}$ is mounted at the end of a set of a two mass-less connected rods of equal length $L=1 \text{ m}$ as shown in Fig .4-b. .If the torsion rigidity of the first rod $GJ_p =1258 \text{ N.m}^2$ and the flexural rigidity of the second rod $EI =1625 \text{ N.m}^2$.Design the proper dynamic absorber such that the mass ratio

$$\frac{m_2}{m_1} = 0.2 .$$

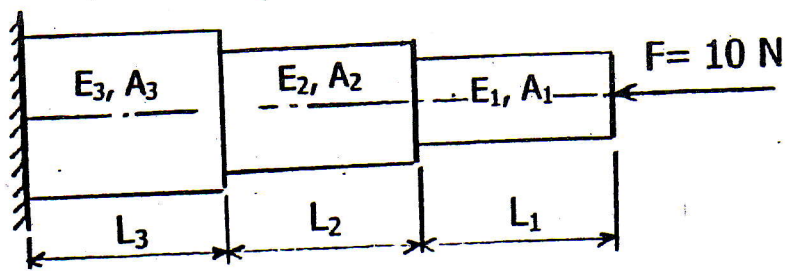


Fig 1-a

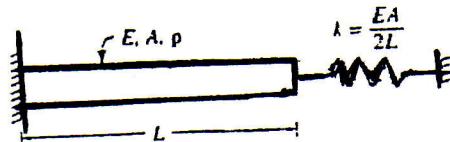


Fig 1-b

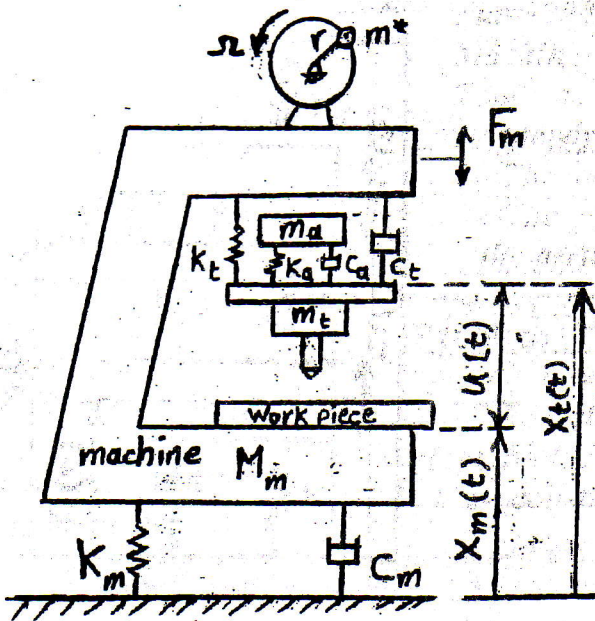


Fig - 2

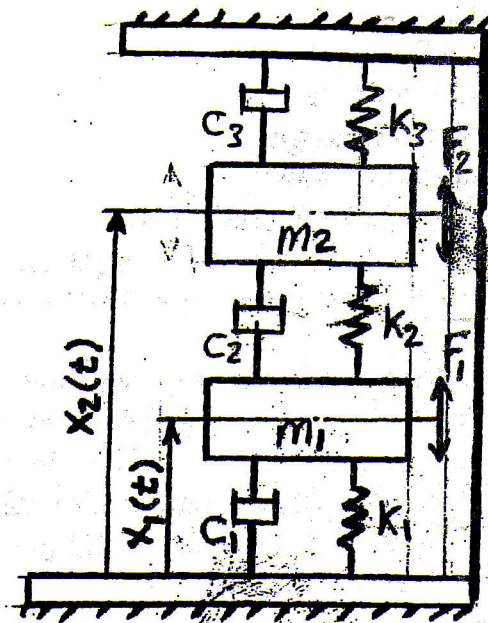


Fig- 3

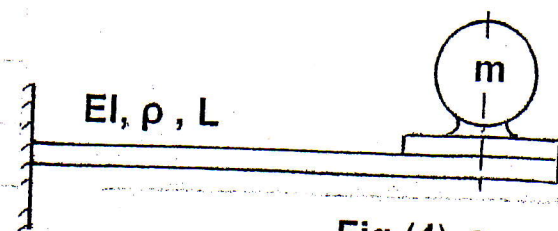


Fig.(4).a

